

### Tutorial 1

1. The strain energy density  $W$  during the deformation can be calculated from

$$W = \int_0^{\epsilon_{ij}} \sigma_{ij} d\epsilon_{ij}. \quad (1)$$

Assume that the elastic part of deformation can be ignored, prove that

$$W = \int_0^{\epsilon_{ij}} \sigma'_{ij} d\epsilon_{ij}. \quad (2)$$

where  $\sigma'_{ij}$  is the deviatoric stress.

Solution:

Elastic deformation is ignored  $\Rightarrow d\epsilon_{ij} = d\epsilon_{ij}^p$ .

According to plastic flow rule,  $d\epsilon_{ij}^p = \sigma'_{ij} d\lambda$ .

Therefore,  $W = \int_0^{\epsilon_{ij}} \sigma_{ij} d\epsilon_{ij} = \int_0^{\epsilon_{ij}} \sigma_{ij} \sigma'_{ij} d\lambda$ .

Since  $\sigma_{ij} \sigma'_{ij} = (\sigma'_{ij} + \sigma_H \delta_{ij}) \sigma'_{ij} = \sigma'_{ij} \sigma'_{ij} + \sigma_H \delta_{ij} \sigma'_{ij} = \sigma'_{ij} \sigma'_{ij}$ , one has

$$W = \int_0^{\epsilon_{ij}} \sigma'_{ij} \sigma'_{ij} d\lambda = \int_0^{\epsilon_{ij}} \sigma'_{ij} d\epsilon_{ij}^p = \int_0^{\epsilon_{ij}} \sigma'_{ij} d\epsilon_{ij}.$$

2. Show that the equivalent stress of a stress tensor  $\sigma_{ij}$  can be expressed as

$$\bar{\sigma} = \frac{1}{\sqrt{2}} \sqrt{(\sigma_{11} - \sigma_{22})^2 + (\sigma_{22} - \sigma_{33})^2 + (\sigma_{33} - \sigma_{11})^2 + 6(\sigma_{12}^2 + \sigma_{23}^2 + \sigma_{13}^2)}. \quad (3)$$

Solution:

The deviatoric stress tensor is

$$\boldsymbol{\sigma}' = \begin{bmatrix} \frac{2\sigma_{11} - \sigma_{22} - \sigma_{33}}{3} & \sigma_{12} & \sigma_{13} \\ \sigma_{12} & \frac{2\sigma_{22} - \sigma_{11} - \sigma_{33}}{3} & \sigma_{23} \\ \sigma_{13} & \sigma_{23} & \frac{2\sigma_{33} - \sigma_{11} - \sigma_{22}}{3} \end{bmatrix}.$$

$\Rightarrow$

$$\begin{aligned} \bar{\sigma} &= \sqrt{\frac{3}{2} \sigma'_{ij} \sigma'_{ij}} \\ &= \sqrt{\frac{3}{2} \left[ \left( \frac{2\sigma_{11} - \sigma_{22} - \sigma_{33}}{3} \right)^2 + \left( \frac{2\sigma_{22} - \sigma_{11} - \sigma_{33}}{3} \right)^2 + \left( \frac{2\sigma_{33} - \sigma_{11} - \sigma_{22}}{3} \right)^2 + 2\sigma_{12}^2 + 2\sigma_{13}^2 + 2\sigma_{23}^2 \right]}. \end{aligned}$$

3. Show that the von Mises yield criterion can be expressed as

$$\sigma_1'^2 + \sigma_2'^2 + \sigma_3'^2 = 2\tau_Y^2, \quad (4)$$

where  $\sigma_1'$ ,  $\sigma_2'$  and  $\sigma_3'$  are principal stresses of the deviatoric stresses tensor  $\sigma'_{ij}$ , and  $\tau_Y$  is the yield shear stress.

Solution:

In the coordinate system with axes along principal directions, the deviatoric tensor can be

written as  $\begin{bmatrix} \sigma_1' & 0 & 0 \\ 0 & \sigma_2' & 0 \\ 0 & 0 & \sigma_3' \end{bmatrix}$ . Therefore equivalent stress is  $\bar{\sigma} = \sqrt{\frac{3}{2} \sigma'_{ij} \sigma'_{ij}} = \sqrt{\frac{3}{2} (\sigma_1'^2 + \sigma_2'^2 + \sigma_3'^2)}$ .

In the coordinate system with axis 1 along the torsion axis, the deviatoric tensor at the yielding

state can be written as  $\begin{bmatrix} 0 & \tau_Y & 0 \\ \tau_Y & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$ . Therefore the equivalent stress is  $\bar{\sigma} = \sqrt{\frac{3}{2} \sigma'_{ij} \sigma'_{ij}} = \sqrt{3} \tau_Y$ .

Equivalent stress is a variable independent of the coordinate system chosen, therefore the two

equivalent stress should be the same, i.e.,  $\sqrt{\frac{3}{2} (\sigma_1'^2 + \sigma_2'^2 + \sigma_3'^2)} = \sqrt{3} \tau_Y$ .

4. Determine the final yield stress of a billet, made from a material with an initial stress-strain curve  $\sigma = 18 + 10\varepsilon^{1/2}$ , that has been compressed uniaxially by 25% and then extended uniaxially by 25% of its intermediate length.

Solution:

The uniaxial tensile test gives the initial (i.e., the equivalent plastic strain is zero at the beginning) true stress-strain relation as  $\sigma = (18 + 10\varepsilon^{1/2})$  MPa. In this curve, the elastic stage is assumed as rigid.

This curve gives the relation between the equivalent stress  $\bar{\sigma}$  and the equivalent plastic strain  $\bar{\varepsilon}^p$  as  $\bar{\sigma} = 18 + 10\sqrt{\bar{\varepsilon}^p}$ .

The final equivalent strain consists of two parts,  $\bar{\varepsilon}^p = \bar{\varepsilon}_1^p + \bar{\varepsilon}_2^p$ . Part 1 (25% compression) gives

$$\bar{\varepsilon}_1^p = \left| \ln \frac{3}{4} \right| = -\ln \frac{3}{4}. \text{ Part 2 (25\% tension) gives } \bar{\varepsilon}_2^p = \ln \frac{5}{4}. \text{ Therefore, } \bar{\varepsilon}^p = -\ln \frac{3}{4} + \ln \frac{5}{4} = \ln \frac{5}{3}.$$

The equivalent stress  $\bar{\sigma} = \left(18 + 10\sqrt{\ln\frac{5}{3}}\right)$  MPa. In uniaxial tension the equivalent stress and the uniaxial stress are the same, therefore, the final yield stress is  $\left(18 + 10\sqrt{\ln\frac{5}{3}}\right)$  MPa

5. The deformation of a metal element is described by the strain rate tensor

$$\dot{\boldsymbol{\varepsilon}} = \begin{bmatrix} 2 & 0 & \sqrt{2} \\ 0 & -3 & 0 \\ \sqrt{2} & 0 & 1 \end{bmatrix} \times 10^{-3} \text{s}^{-1}. \text{ Assuming that elastic strains are negligible (Levy-Mises flow}$$

rule can be applied), determine:

- (i) The principal strain rate;
- (ii) The deviatoric stress tensor  $\boldsymbol{\sigma}'$ , where the maximum principal deviatoric stress for the element is 100MPa;
- (iii) The associated stress tensor for the element, where its mean stress is 100MPa;
- (iv) The unit normals for the surfaces on which the principal stresses act.

Solution:

- (i) Denote the principal strain rate as  $\lambda \times 10^{-3} \text{s}^{-1}$ . Solving the equation

$$\begin{vmatrix} 2-\lambda & 0 & \sqrt{2} \\ 0 & -3-\lambda & 0 \\ \sqrt{2} & 0 & 1-\lambda \end{vmatrix} = 0$$

$$\Rightarrow \begin{matrix} \lambda_1 = 3 \\ \lambda_2 = 0 \\ \lambda_3 = -3 \end{matrix}, \text{ so the principal strain rates are } \begin{cases} \dot{\varepsilon}_1 = 3 \times 10^{-3} \text{s}^{-1} \\ \dot{\varepsilon}_2 = 0 \\ \dot{\varepsilon}_3 = -3 \times 10^{-3} \text{s}^{-1} \end{cases}.$$

- (ii) In the principal coordinate system, the strain rate tensor is  $\begin{bmatrix} \dot{\varepsilon}_1 & 0 & 0 \\ 0 & \dot{\varepsilon}_2 & 0 \\ 0 & 0 & \dot{\varepsilon}_3 \end{bmatrix}$ . According to

the flow rule The deviatoric stress tensor is  $\begin{bmatrix} \dot{\varepsilon}_1 & 0 & 0 \\ 0 & \dot{\varepsilon}_2 & 0 \\ 0 & 0 & \dot{\varepsilon}_3 \end{bmatrix} \frac{1}{\lambda}$ . Since  $\frac{\varepsilon_1}{\lambda} = 100 \text{MPa}$ ,

$$\dot{\lambda} = 3 \times 10^{-5} \frac{1}{\text{MPa} \cdot \text{s}}.$$

Therefore in the principal coordinate system, the deviatoric stress tensor is

$$100 \times \begin{bmatrix} 1 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & -1 \end{bmatrix} \text{MPa}.$$

In the original coordinate system (for strain rate tensor  $\dot{\epsilon} = \begin{bmatrix} 2 & 0 & \sqrt{2} \\ 0 & -3 & 0 \\ \sqrt{2} & 0 & 1 \end{bmatrix} \times 10^{-3} \text{s}^{-1}$ ), the

deviatoric stress tensor can be expressed as  $\frac{100}{3} \times \begin{bmatrix} 2 & 0 & \sqrt{2} \\ 0 & -3 & 0 \\ \sqrt{2} & 0 & 1 \end{bmatrix} \text{MPa}.$

(iii) The hydrostatic stress (mean stress) is 100MPa.

In the principal coordinate system, the stress tensor is

$$\left( \begin{bmatrix} 1 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & -1 \end{bmatrix} + \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \right) \times 100 \text{MPa} = \begin{bmatrix} 2 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 0 \end{bmatrix} \times 100 \text{MPa}.$$

In the original coordinate system, the deviatoric stress tensor can be expressed as

$$\left( \begin{bmatrix} \frac{2}{3} & 0 & \frac{\sqrt{2}}{3} \\ 0 & -1 & 0 \\ \frac{\sqrt{2}}{3} & 0 & \frac{1}{3} \end{bmatrix} + \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \right) \times 100 \text{MPa} = \begin{bmatrix} \frac{5}{3} & 0 & \frac{\sqrt{2}}{3} \\ 0 & 0 & 0 \\ \frac{\sqrt{2}}{3} & 0 & \frac{4}{3} \end{bmatrix} \times 100 \text{MPa}$$

(v) Finding the unit vector of the principal directions:  $\underline{n}_1, \underline{n}_2$  and  $\underline{n}_3$ .

1<sup>st</sup> principal direction

$$\begin{bmatrix} -1 & 0 & \sqrt{2} \\ 0 & -6 & 0 \\ \sqrt{2} & 0 & -2 \end{bmatrix} \begin{pmatrix} n_1 \\ n_2 \\ n_3 \end{pmatrix}_1 = \begin{pmatrix} 0 \\ 0 \\ 0 \end{pmatrix} \Rightarrow n_2 = 0, n_1 = \sqrt{2}n_3 \text{ and } n_1^2 + n_2^2 + n_3^2 = 1 \Rightarrow n_3 = \frac{1}{\sqrt{3}}$$

$$\underline{n}_1 = \begin{pmatrix} n_1 \\ n_2 \\ n_3 \end{pmatrix}_1 = \frac{1}{\sqrt{3}} \begin{pmatrix} \sqrt{2} \\ 0 \\ 1 \end{pmatrix}$$

2nd principal direction

$$\begin{bmatrix} 2 & 0 & \sqrt{2} \\ 0 & -3 & 0 \\ \sqrt{2} & 0 & 1 \end{bmatrix} \begin{pmatrix} n_1 \\ n_2 \\ n_3 \end{pmatrix}_2 = \begin{pmatrix} 0 \\ 0 \\ 0 \end{pmatrix} \Rightarrow n_2 = 0, n_3 = -\sqrt{2}n_1 \text{ and } n_1^2 + n_2^2 + n_3^2 = 1 \Rightarrow n_1 = \frac{1}{\sqrt{3}}$$

$$\underline{n}_2 = \begin{pmatrix} n_1 \\ n_2 \\ n_3 \end{pmatrix}_2 = \frac{1}{\sqrt{3}} \begin{pmatrix} 1 \\ 0 \\ -\sqrt{2} \end{pmatrix},$$

3rd principal direction

$$\begin{bmatrix} 5 & 0 & \sqrt{2} \\ 0 & 0 & 0 \\ \sqrt{2} & 0 & 4 \end{bmatrix} \begin{pmatrix} n_1 \\ n_2 \\ n_3 \end{pmatrix}_3 = \begin{pmatrix} 0 \\ 0 \\ 0 \end{pmatrix} \Rightarrow \underline{n}_3 = \begin{pmatrix} n_1 \\ n_2 \\ n_3 \end{pmatrix}_3 = \begin{pmatrix} 0 \\ 1 \\ 0 \end{pmatrix}$$